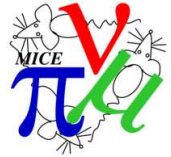




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# Integration of Cavities and Coupling Coil Modules

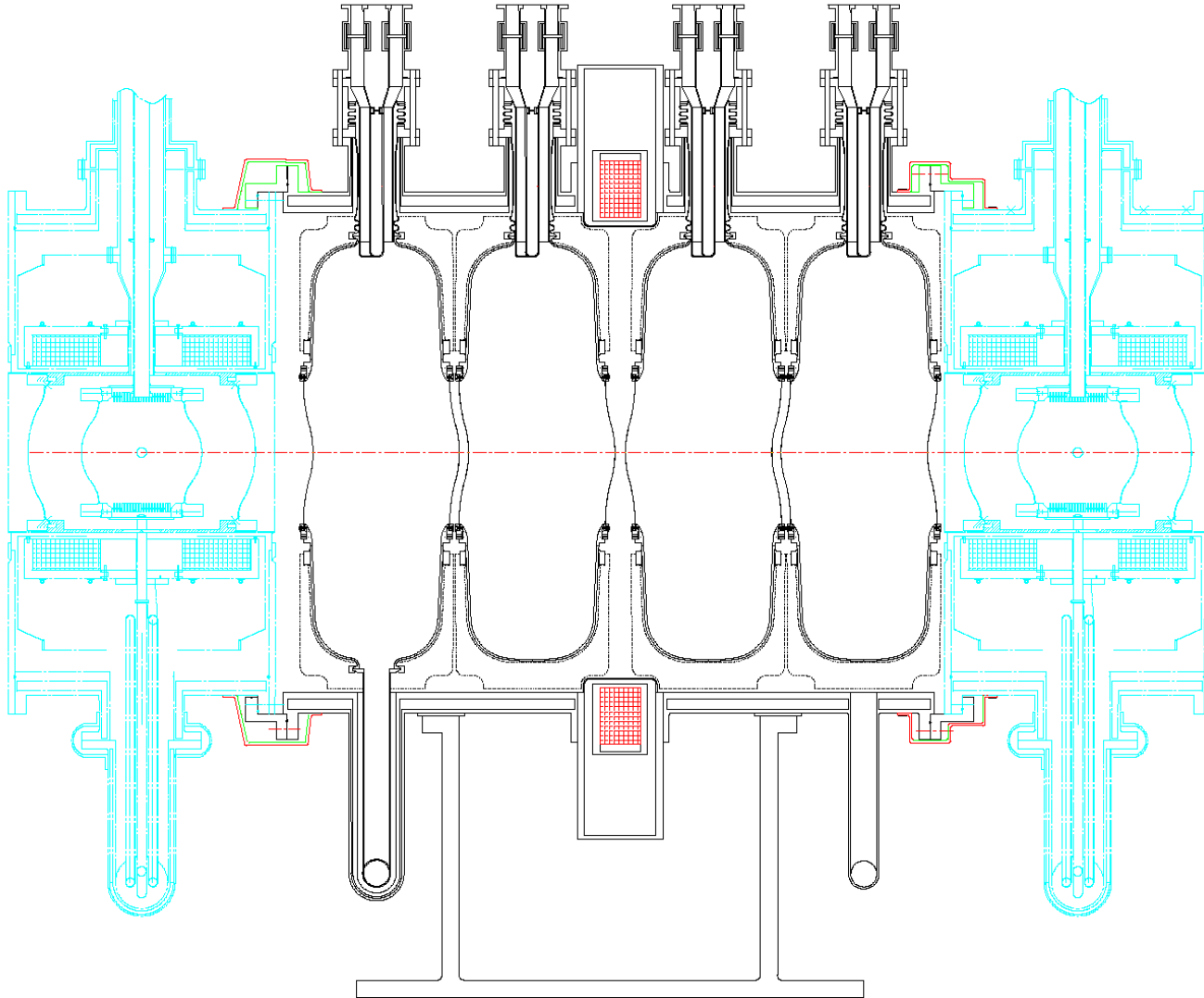
MICE Collaboration Meeting

November 2, 2003

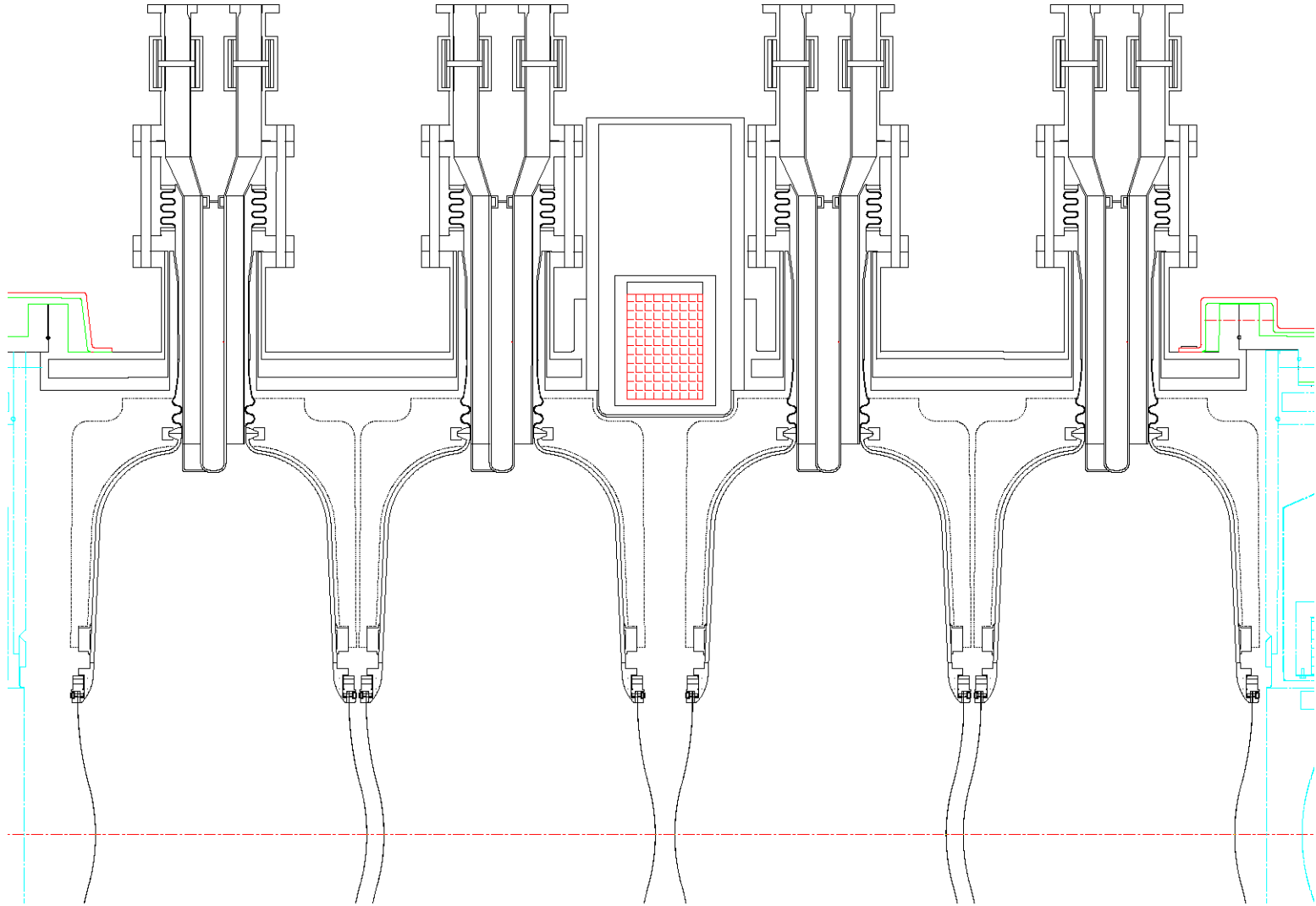
**Steve Virostek**

Lawrence Berkeley National Laboratory

# Cavity/Coil Module Layout

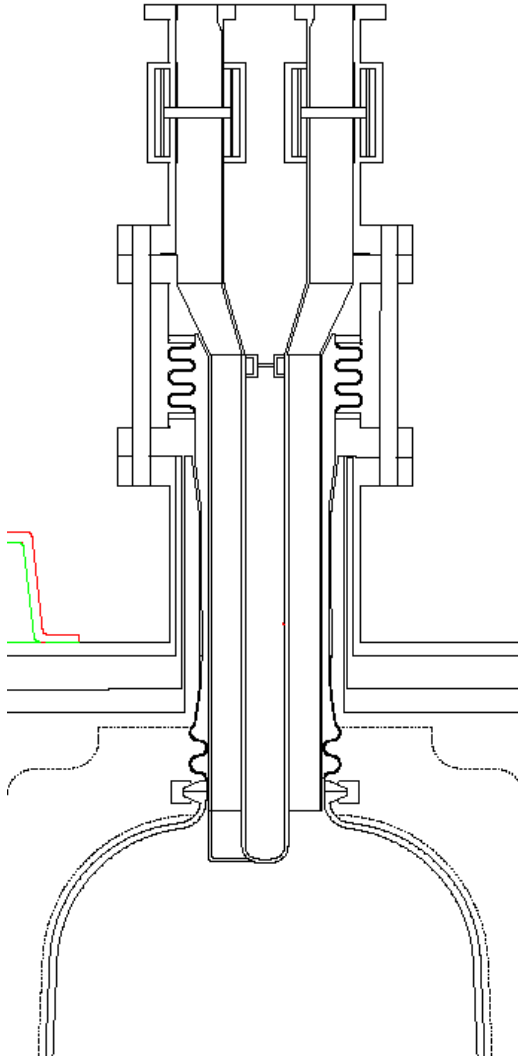


# Close-up View of Module



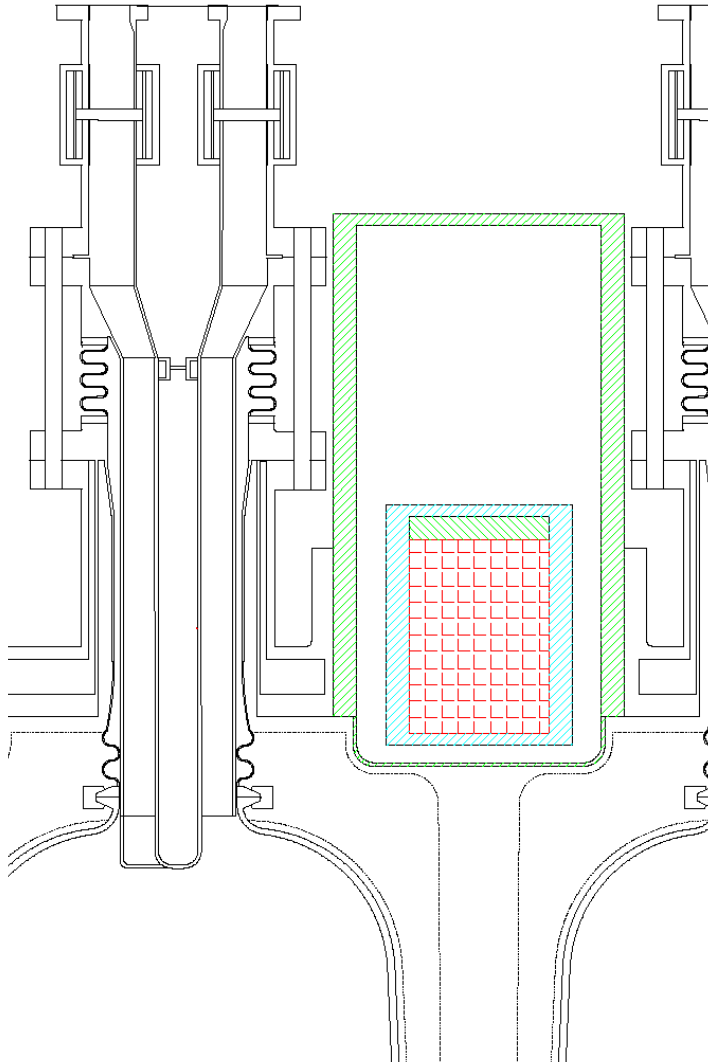
- RF coupler configuration
- Coupling coil geometry
- Cavity tuning mechanism
- Cavity mounting scheme
- Cavity port connections
- Vacuum vessel flange joints
- Module assembly scheme

# RF Coupler Configuration



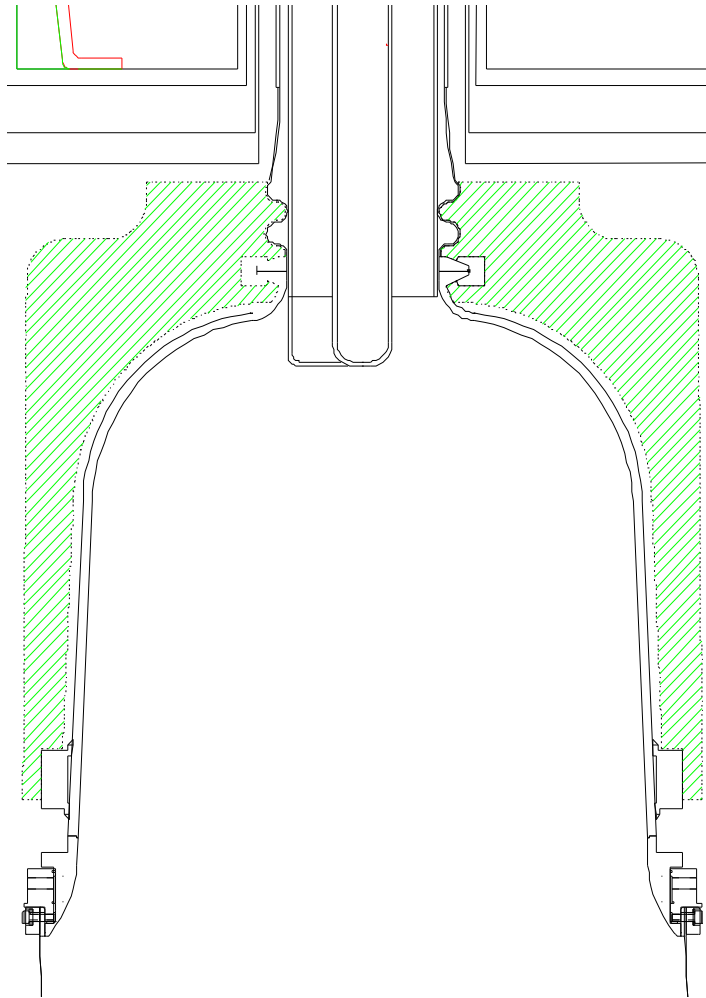
- Proposed layout allows all couplers to be oriented normal to vacuum shell
- All cavities are identical
- All couplers are identical
- Normal orientation facilitates cavity alignment and installation
- Tolerance stack-up effect reduced
- Cavities, couplers & vacuum shell are simpler and less expensive

# Coupling Coil Geometry



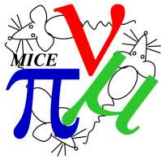
- Revised coil design (per Mike Green) is much narrower than the previous design
- New design allows normal coupler geometry and increases interior clearance for tuners
- Minimal impact on cost of coupling coil fabrication

# Cavity Tuning Mechanism



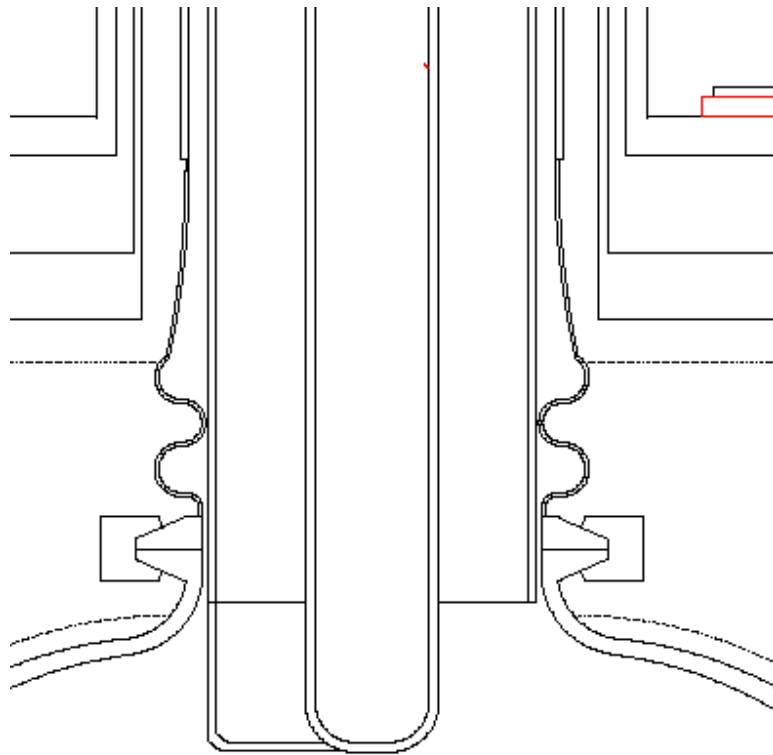
- Finite element analysis of cavity stiffness, stress and frequency sensitivity is complete
- Six discrete tuning mechanisms mounted on each cavity
- Tuners synchronized through control of actuators
- Conceptual design in progress
- Envelope defined as shown

# Cavity Mounting Scheme



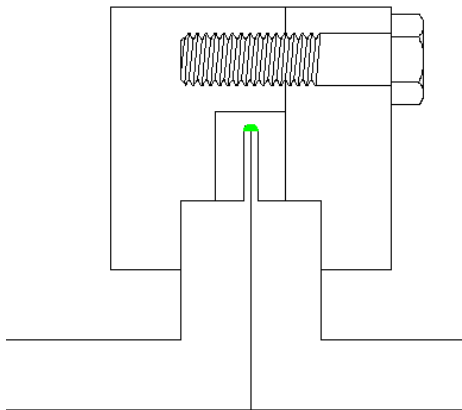
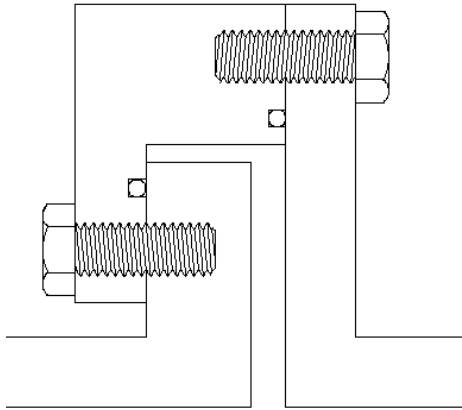
- No interconnection between adjacent cavities
- Cavities are mounted to vacuum vessel walls through tuner mechanisms
- Cavity centers do not shift during tuning
- RF coupler and vacuum cavity connections can be rigid
- Allows a much simpler approach to module assembly

# Cavity Port Connections



- All ports are on the cavity equator
- Local cavity shape is spherical to facilitate port construction
- Flange clamp applied after cavity is installed in vacuum shell
- Perfect vacuum seal not required
- Good RF contact will be achieved by using a spring ring or a lip seal

# Vacuum Vessel Flange Joints



- O-ring or tin seal flange design allows removal without dragging of seal surface
- O-ring seal is less reliable than other options and adds to vessel vacuum load
- Tin seal has advantages of O-ring without gas load or limited lifetime issues
- Welded joint provides a clean, reliable seal
- Small weld bead allows easier removal
- Clamping flange provides strength
- Welded flange doesn't allow gap and limits the number of removals

# Module Assembly Scheme



- Curved beryllium windows are installed on cavities
- Tuner mechanisms are applied to cavities
- Coupling coil can is welded to module vacuum vessels
- Assembled cavities are inserted into vacuum vessel from the ends, one at a time
- Tuner mechanisms are connected to vessel wall
- RF couplers are installed and flange clamps are applied
- Vacuum, cooling and electrical connections are made

# LN2 Operation Issues

- Frequency shift caused by thermal contraction
- Thermal losses to vacuum vessel through cavity mounts and other connections
- Mechanical stresses at connections between cavity and vessel (tuner mounts, RF couplers, vacuum, etc.) caused by thermal contraction of cavity

## Possible Solutions:

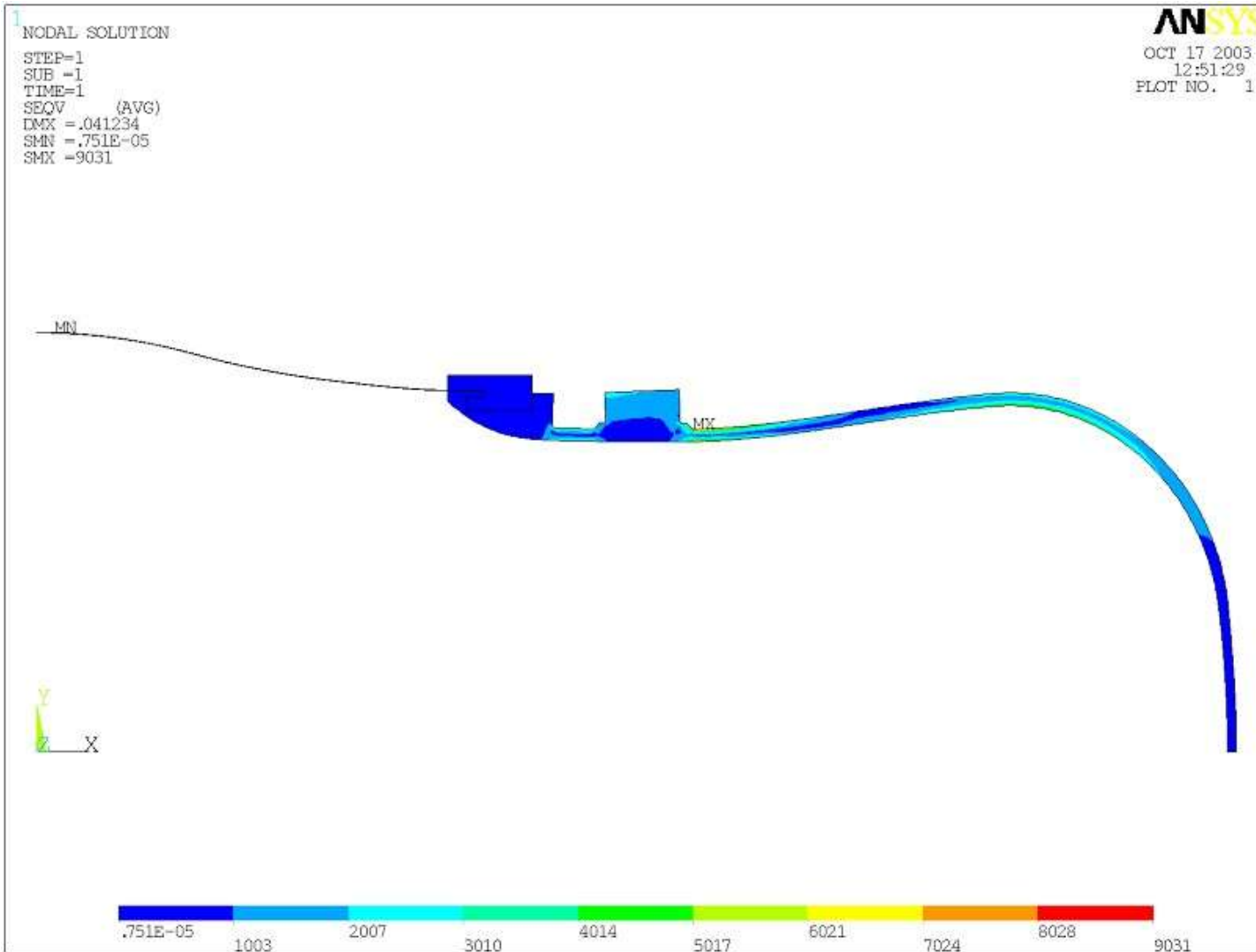
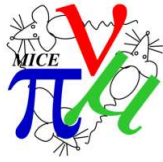
- Apply force on tuner rings to permanently deform cavity & lower frequency
- Use low thermal conductivity isolator material for cavity-vessel connections
- Design flexibility in cavity connections to allow thermal contraction

# Cavity Stiffness Analysis



- An analysis was carried out to determine the response of the cavity to tuning and vacuum loads
- A 2D axisymmetric structural model (ANSYS) was used to calculate the cavity stresses, forces and deformed shape when subjected to tuner ring displacements
- The displaced shape for each case was fed into an ANSYS RF model to determine the resonant frequency
- Comparison to results for the undeformed cavity gives the net frequency shift
- Parameters to be used for tuner mechanism design

# Cavity Finite Element Analysis



# Summary of Cavity Analysis

Ring Displ. (mm)	Freq. Shift (kHz)	Total Ring Load (lb)	Ring Load per Pt. (lb)	Stress @ Weld (ksi)	Next Highest Stress (ksi)
+1	+228	2,750	458	17.7	9.3
+2	+456	5,500	917	35.4	18.6
-1	-230	-2,750	-458	17.7	9.3
-2	-460	-5,500	-917	35.4	18.6

Frequency Sensitivity: 229 kHz/mm @ stiffener ring

Cavity Stiffness: 2750 lb/mm @ stiffener ring